THE THERMAL BOUNDARY LAYER OF A NON-FOURIER POWER-LAW FLUID ON A PLATE WITH A VARIABLE SURFACE TEMPERATURE

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The equation for the thermal boundary layer of a non-Fourier powerlaw fluid on a flat plate with an exponential distribution of surface temperature is reduced to an ordinary equation and solved by the method of finite differences. The effect of the exponent $\gamma$ on the temperature profile and on the heat-transfer coefficient is determined. It is demonstrated that the asymptotic solutions of the equation for large $\sigma$ are nearly exact.

With consideration of the non-Fourier heat-conduction law [1]

$$
\begin{equation*}
q=-H\left(\frac{\partial u_{1}}{\partial y_{1}}\right)^{n-1} \frac{\partial T}{\partial y_{1}} \tag{1}
\end{equation*}
$$

the equation for the thermal boundary layer, with viscous dissipation neglected, is written in the form

$$
\begin{gather*}
u_{1} \frac{\partial T}{\partial x_{1}} \div v_{1} \frac{\partial T}{\partial y_{1}}=  \tag{2}\\
=\frac{H}{\rho c_{p}} \frac{\partial}{\partial y_{1}}\left[\left(\frac{\partial u_{1}}{\partial y_{1}}\right)^{n-1} \frac{\partial T}{\partial y_{1}}\right] .
\end{gather*}
$$

We assume the following boundary conditions:

$$
\begin{gather*}
T=T_{w}\left(x_{1}\right) \text { when } y_{1}=0 ; \quad T \rightarrow T_{\infty} \text { as } y_{1} \rightarrow \infty ;  \tag{3}\\
T_{w}\left(x_{1}\right)=T_{\infty}+A x_{1}^{\gamma} . \tag{4}
\end{gather*}
$$

Equation (2) is written as follows in dimensionless form:

$$
\begin{align*}
& \gamma \frac{u}{x} \theta+u \frac{\partial \theta}{\partial x}+v \frac{\partial \theta}{\partial y}= \\
& =\frac{1}{\sigma} \frac{\partial}{\partial y}\left[\left(-\frac{\partial u}{\partial y}\right)^{n-1} \frac{\partial \theta}{\partial y}\right] . \tag{5}
\end{align*}
$$

Here

$$
u=\frac{u_{1}}{U}, \quad v=\frac{v_{1}}{U} \mathrm{R}^{\frac{1}{1+n}}
$$

$$
\begin{gather*}
x=\frac{x_{1}}{L}, \quad y=\frac{y_{1}}{L} \mathrm{R}^{\frac{1}{1+n}}, \\
\mathrm{R}=\frac{\rho U^{2}-n L^{n}}{K}, \\
\theta=\frac{T-T_{\infty}}{T_{w}^{*}\left(x_{1}\right)-T_{\infty}}, \quad \sigma=\frac{K c_{p}}{H} . \tag{6}
\end{gather*}
$$

For the boundary layer of a non-Newtonian powerlaw fluid on a flat plate, $u$ and $v$ are determined from the following formulas [2]:

$$
\begin{gather*}
u=\varphi^{\prime}(\eta), v=[n(1+n) x]^{\frac{1}{1+n}}[(1+n) x]^{-1}\left[\eta \varphi^{\prime}-\varphi\right], \\
\eta=y[n(1+n) x]^{-\frac{1}{1+n}}, \tag{7}
\end{gather*}
$$

where the prime denotes the derivative with respect to $\eta$, and the function $\varphi$ satisfies the equation

$$
\begin{equation*}
\varphi^{\prime \prime \prime}+\varphi\left(\varphi^{\prime \prime}\right)^{2-n}=0 \tag{8}
\end{equation*}
$$

with the boundary conditions

$$
\begin{equation*}
\varphi=0, \varphi^{\prime}=0 \text { when } \eta=0 ; \varphi^{\prime} \rightarrow 1 \text { as } \eta \rightarrow \infty \tag{9}
\end{equation*}
$$

With formulas (7), Eq. (5) is reduced to an ordinary differential equation

$$
\begin{gather*}
\theta^{\prime \prime}+\left[(n-1) \varphi^{\prime \prime \prime} / \varphi^{\prime \prime}+\sigma n\left(\varphi^{\prime \prime}\right)^{1-n} \varphi\right] \theta^{\prime}- \\
-\sigma n(1+n) \gamma \varphi^{\prime}\left(\varphi^{\prime \prime}\right)^{1-n} \theta=0 \tag{10}
\end{gather*}
$$

or with consideration of (8)

$$
\begin{gather*}
\theta^{\prime \prime}+[(\sigma-1) n+1]\left(\varphi^{\prime \prime}\right)^{1-n} \varphi \theta^{\prime}- \\
-\sigma n(1+n) \gamma \varphi^{\prime}\left(\varphi^{\prime \prime}\right)^{1-n} \theta=0 \tag{11}
\end{gather*}
$$

Table 1
The Values of $-\theta^{\prime}(0)$

| $n$ | $\gamma$ |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | 0 | -0.5 | 0 | 0.5 | 2.0 | 4.0 |
| 0.5 | 10 | 0.2431 | 0.5549 | 0.7155 | 0.9914 | 1.217 |
|  | 100 | 0.5152 | 1.188 | 1.535 | 2.132 | 2.612 |
| 0.7 | 10 | 2.215 | 0.7555 | 0.9995 | 1.406 | 1.731 |
|  | 100 | 0.4750 | 1.625 | 2.149 | 3.027 | 3.723 |
| 1.0 | 10 | 0.0000 | 1.030 | 1.411 | 2.016 | 2.490 |
|  | 100 | 0.0000 | 2.223 | 3.041 | 4.345 | 5.364 |
| 1.5 | 10 | $-1.466$ | 1.396 | 2.012 | 2.930 | 3.633 |
|  | 100 | $-3.331$ | 3.021 | 4.342 | 6.311 | 7.829 |
| 2.0 | 10 | -29.31 | 1.662 | 2.503 | 3.692 | 4.591 |
|  | 100 | -567.0 | 3.602 | 5.404 | 7.958 | 9.890 |

The boundary condition (3) will be

$$
\begin{equation*}
\theta=1 \text { when } \eta=0 ; \theta \rightarrow 0 \text { as } \eta \rightarrow \infty . \tag{12}
\end{equation*}
$$

Levy [3] dealt with an analogous problem for a Newtonian fluid and an arbitrary surface; the problem was formulated in [2] for a power-law fluid and for Eq. (1).

To solve the 2 -nd order linear equation (11), we employ the method of finite differences used in [3], with slight modifications.

We will divide the region of integration into equal intervals of length $h$ and, replacing the derivatives $\theta_{\mathrm{i}}^{\prime \prime}$ and $\theta_{\mathrm{i}}^{\prime}$ by the finite-difference ratios, instead of (11) we obtain

$$
\begin{gather*}
\left(\theta_{i+1}-2 \theta_{i}+\theta_{i-1}\right) / h^{2}+ \\
+A_{i}\left(\theta_{i+1}-\theta_{i-1}\right) / h-B_{i} \theta_{i}=0 \tag{13}
\end{gather*}
$$

where

$$
\begin{gather*}
A_{i}=[(\sigma-1) n+1]\left(\varphi_{i}^{\prime \prime}\right)^{1-n} \varphi_{i} / 2 \\
\quad B_{i}=\sigma n(1+n) \gamma \varphi_{i}^{\prime}\left(\varphi_{i}^{\prime \prime}\right)^{1-n} \tag{14}
\end{gather*}
$$

This yields the formula

$$
\begin{equation*}
\theta_{i+1}=a_{i} \theta_{i}-b_{i} \theta_{i-1} \tag{15}
\end{equation*}
$$

when

$$
\begin{align*}
a_{i} & =\left(2+h^{2} B_{i}\right) /\left(1+h A_{i}\right), \\
b_{i} & =\left(1-h A_{i}\right) /\left(1+h A_{i}\right) \tag{16}
\end{align*}
$$

Using formula (15) for $\mathbf{i}=1,2,3$, etc., with consideration of the boundary condition $\theta_{0}=1$, which fol-


Fig. 1. Temperature profiles in the boundary layer at $\mathrm{n}=0.5 ; \sigma=10$ : 1) $\gamma=-0.75$; 2) $-2 / 3=\gamma_{0}$; 3) -0.5 ; 4) 0 ; 5) 0.5 ; 6) 2.0 ; 7) 4.0 .
lows from (12), we find that

$$
\begin{equation*}
\theta_{i+1}=c_{i} \theta_{1}-d_{i} \tag{17}
\end{equation*}
$$

where

$$
\begin{equation*}
c_{i}=a_{i} c_{i-1}-b_{i} c_{l-2} ; \quad d_{i}=a_{i} d_{l-1}-b_{i} d_{i-2} \tag{18}
\end{equation*}
$$

and

$$
\begin{gather*}
c_{1}=a_{1}, \quad c_{0}=1, \quad c_{-1}=0 \\
d_{1}=b_{1}, \quad d_{0}=0, \quad d_{-1}=-1 \tag{19}
\end{gather*}
$$

Applying the second boundary condition of (12) in the form of $\theta_{\mathrm{r}+1}=0$ to formula (17), with $\mathbf{r}$ correspond-
ing to the external edge of the boundary layer, we have

$$
\begin{equation*}
\theta_{1}^{(r)}=d_{r} / c_{r} . \tag{20}
\end{equation*}
$$

The value of $i=r$ is obtained from the condition

$$
\begin{equation*}
\left|\theta_{l}^{(i)}-\theta_{1}^{(i-1)}\right| / h<\varepsilon, \tag{21}
\end{equation*}
$$

where $\varepsilon$ is a small number specified in advance. This


Fig. 2. Temperature profiles in boundary layer at $\mathrm{n}=1.0 ; \sigma=10 ; 1) \gamma=-0.6$; 2) $-0.5=\gamma_{0}$; 3) 0 ; 4) 0.5 ; 5) 2.0 ; 6) 4.0.
corresponds to the approach of the ratio $\mathrm{d}_{\mathrm{i}} / \mathrm{c}_{\mathbf{i}}$ to some limit determining $\theta_{1}$ in Eq. (20).

As a result, the solution of the problem is obtained with formulas (14) and (16)-(21). Initially the coefficients $c_{i}$ and $d_{i}$ are calculated according to formulas (18)-(19); the value of $\theta_{1}$ from Eq. (20) is found, and then the profile $\theta$ from (17) is calculated. The solution was obtained on a Ural-2 computer; the solution of Eq. (8), required for the calculation of the coefficients in (14), was found by the Runge-Kutta method with consideration of the values of $\varphi^{9 \prime}(0)$, which are known from [2].

The quantity $-\theta^{\prime}(0)$, shown in Table 1, is determined from the formula

$$
\begin{equation*}
-\theta^{\prime}(0)=\left(1-\theta_{1}\right) / h . \tag{22}
\end{equation*}
$$

Comparison of the values of $\theta^{\prime}(0)$ with the corresponding quantities calculated by other methods for $\gamma=0[2,4]$ demonstrated that the described method, given proper selection of $h$ and $\varepsilon$, yields no less than 4 exact significant figures. Good agreement with the results of [3] was not noted, although this can be explained by the fact that the accuracy of the difference equation corresponding to (13) is lower by an order of magnitude in [3] than in this paper [ $\mathrm{O}(\mathrm{h})$ instead of $\left.O\left(h^{2}\right)\right]$, since a unilateral formula was used in [3] for the derivative $\theta_{\mathrm{i}}^{?}$ (in (13) a central formula was used) and the calculations were carried out for a constant and rather large pitch of $h=0.1$. We note that in the calculation of $\theta^{\prime}(0)$ in accordance with the unilateral formula (22) there is no loss of accuracy, since $\theta^{\prime \prime}(0)=0$.

Examples of the calculated $\theta(\eta)$ profiles are shown in Figs. 1-3. We see that an increase in $\gamma$ results in a refinement of thermal boundary layer. When $\gamma<0$, the profiles exhibit a flexure point within the layer,

Table 2

$$
\text { Comparison of the Exact and Approximate Values of }-\theta^{9}(0)
$$

| $\beta$ | $n$ | $\sigma$ |  |  |  |  |  | $\gamma$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  | 10 |  |  | 100 |  |  |  |
|  |  | exact | $\begin{gathered} \text { formula } \\ (33) \\ \hline \end{gathered}$ | error, \% | exact | $\begin{gathered} \text { formula } \\ (33) \end{gathered}$ | error, |  |
| 0.2735 | 0.5 | 0.6691 | 0.6664 | -0.40 | 1.433 | 1.436 | +0.21 | 0.3333 |
| 0.3567 | 0.7 | 0.9128 | 0.9121 | -0.08 | 1.963 | 1.965 | +0.10 | 0.2941 |
| 0.4696 | 1.0 | 1.246 | 1.248 | +0.16 | 2.687 | 2.689 | +0.07 | 0.25 |
| 0.6189 | 1.5 | 1.691 | 1.697 | $+0.35$ | 3.653 | 3.656 | $+0.08$ | 0.2 |
| 0.7265 | 2.0 | 2.014 | 2.023 | +0.45 | 4.356 | 4.358 | +0.05 | 0.1667 |

while for some $\gamma=\gamma_{0}$ the condition $\theta^{\prime}(0)=0$ is satisfied. When $\gamma<\gamma_{0}, \theta^{\prime}(0)>0$ and the temperature within the boundary layer exceeds the wall temperature ( $\theta>$ $>1$ ).


Fig. 3. Temperature profiles in the boundary layer at $n=1.5 ; \sigma=10$ :

$$
\text { 1) } \gamma=-0.5 \text {; 2) }-0.4=\gamma_{0} \text {; 3) } 0 \text {; }
$$

4) 0.5 ; 5) 2.0 ; 6) 4.0 .

Using formulas (1), (4), (6), and (7), we determine the local heat-transfer coefficient of the plate:

$$
\begin{align*}
& \mathrm{Nu} / \mathrm{R}=-q_{w} K / A L^{\gamma} H \rho U=E x_{1}^{\gamma} R_{x_{1}}^{-\frac{1}{1+n}}  \tag{23}\\
& E=-[n(1+n)]^{-\frac{n}{1+n}}\left[\varphi^{\prime \prime}(0)\right]^{n-1} \theta^{\prime}(0) \tag{24}
\end{align*}
$$

It follows from formula (23) that when $\gamma=\mathrm{n} /(1+\mathrm{n})$, $q_{W}$ is independent of $x_{1}$.

In view of the fact that the problem is solved for $\sigma \gg 1$, the thermal boundary layer is much thinner than the dynamic boundary layer and the following relationships of [4] are approximately valid:

$$
\begin{equation*}
\varphi^{\prime \prime}=\text { const }=\beta, \quad \varphi^{\prime \prime \prime}=0, \quad \varphi^{\prime}=\beta \eta, \quad \varphi=\beta \eta^{2} / 2 . \tag{25}
\end{equation*}
$$

Having substituted (25) into (10), we obtain the equation

$$
\begin{equation*}
\theta^{\prime \prime}+a \eta^{2} \theta^{\prime}+b \eta \theta=0 \tag{26}
\end{equation*}
$$

where

$$
\begin{equation*}
a=\sigma n \beta^{2-n} / 2, \quad b=-\sigma n(1+n) \gamma \beta^{2-n} . \tag{27}
\end{equation*}
$$

In the general case, Eq. (16), with substitution of the variables

$$
\begin{equation*}
\theta=\eta^{-1} \exp \left(-a \eta^{3} / 6\right) W, \quad z=a \eta^{3} / 3 \tag{28}
\end{equation*}
$$

can be transformed into the Whittaker equation [5]

$$
\begin{equation*}
\frac{d^{2} W}{d z^{2}}+\left(-\frac{1}{4}+\frac{k}{z}+\frac{1 / 4-m^{2}}{z^{2}}\right) W=0 \tag{29}
\end{equation*}
$$

when $\mathrm{k}=(\mathrm{b}-a) / 3 a, \mathrm{~m}=1 / 6$, whose solution is expressed in the form of series in powers of $z$ or in contour integrals with the parameter $z$. These solutions are complex for the calculations.

For certain relationships between b and $a$, Eq. (26) is easily solved under the conditions of (12). Thus, when $\mathrm{b}=2 a$ we have the solution

$$
\begin{equation*}
\theta=\exp \left(-a \eta^{3} / 3\right) \tag{30}
\end{equation*}
$$

which corresponds to the case $\theta^{\prime}(0)=0$ and therefore defines the quantity $\gamma_{0}$. It is precisely when $b=2 a$ that it follows from (27) that

$$
\begin{equation*}
\gamma_{0}=-1 /(1+n) . \tag{31}
\end{equation*}
$$

Calculations have shown that formula (31) for the values of $\sigma$ and $n$ under consideration within the limitations of 4 significant figures is exact, while the profiles of (30) virtually do not differ from the results of the numerical solution.

Another simple solution is derived for $b=-a$, i. e. , when $\gamma=1 / 2(1+n)$, and this solution is in the form

$$
\begin{equation*}
\theta=\exp \left(-a \eta^{3} / 3\right)-a \eta \int_{\eta}^{\infty} \eta \exp \left(-a \eta^{3} / 3\right) d \eta, \tag{32}
\end{equation*}
$$

whence

$$
\begin{equation*}
-\theta^{\prime}(0)=(\alpha / 3)^{1 / 3} \Gamma(2 / 3) \cong 0.7452\left(\sigma n \beta^{2-n}\right)^{1 / 3} \tag{33}
\end{equation*}
$$

The accuracy of formula (33) is sufficiently high, as can be seen from Table 2.

The solution for the case $b=0(\gamma=0)$ was given in [4] and it was also close to an exact solution.

The asymptotic equation (26) thus yields a solution that is little different from the solution of Eq. (10).

## NOTATION

$q$ is the heat flux in the boundary layer; $q_{W}$ is the same, at the wall; $H$ is the thermal conductivity char-
acteristic; $K$ and $n$ are the rheological characteristics of the fluid; $x_{1}$ is the longitudinal coordinate; $y_{1}$ is the transverse coordinate; $u_{1}$ and $v_{1}$ are the projections of the velocity vector onto the $\mathrm{x}_{1}-$ and $\mathrm{y}_{1}$-axes, respectively; U is the velocity of external flow; L is the characteristic length; $R$ is the Reynolds number; $\rho$ is the fluid density; $c_{p}$ is the specific heat capacity; $A$ and $\gamma$ are the constants (formula (4)); $\sigma$ is the Prandtl number; Nu is the Nusselt number.

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